

Development of a Fast-Response Diamond Calorimeter Heat Transfer Gauge

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A robust fast-response calorimeter heat transfer gauge called the Diamond Heat Transfer Gauge has been developed for use in transient hypersonic ground test facilities. Gauges have been produced using discs of synthetic diamond 150 μm to 325 μm thick as a calorimeter and platinum thin-film resistance temperature detectors to measure temperature rise. The response times of the gauges are dependent on calorimeter thickness and range between 10.6 μs to 49.6 μs . Heat fluxes greater than 100 MWm^{-2} have been measured at the stagnation point of a model in flows with total enthalpies of up to 70 MJkg^{-1} over multiple shots an expansion tunnel. Experimentally measured heat fluxes compare well with measurements from surface abraded coaxial thermocouples gauges and an empirical correlation.

Nomenclature

a	=	ratio of thermal products
α	=	thermal diffusivity, $\text{m}^2 \text{s}^{-1}$
α_R	=	coefficient of temperature with resistance, K^{-1}
e	=	thermal effusivity, $\text{W m}^{-1} \text{K}^{-1} \text{s}^{-0.5}$
q	=	heat flux, W m^{-2}
c_p	=	specific heat capacity at constant pressure, $\text{J kg}^{-1} \text{K}^{-1}$
H	=	enthalpy, J kg^{-1}
k	=	thermal conductivity
l	=	calorimeter thickness, m

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M	=	Mach number
R_0	=	electrical resistance at 0 °C, Ω
U	=	velocity, m s^{-1}
T	=	temperature, K
V	=	voltage, V
ρ	=	density, kg m^{-3}
Subscripts		
∞	=	free-stream
s	=	surface
1	=	top layer, calorimeter
2	=	substrate
l	=	calorimeter-substrate interface

I. Introduction

High enthalpy ground test facilities produce challenging environments in which to make surface heat flux measurements. Ultra-high speed pulse flow wind tunnels capable of producing total enthalpies equivalent to flight and matching binary scaling parameters have test times of the order 10 μs to 2 ms[1]. Instrumentation used must have sufficiently fast response times to produce useful data while also surviving both the extreme heat rates and impacts from fragments of metallic and plastic diaphragms that impact models at high speeds.

Global heat transfer measurement techniques have improved recently. Fast-response Temperature Sensitive Paints (TSP) have been developed but has drawbacks including: limited durability in high enthalpy facilities (perhaps only lasting one tunnel shot)[2], the need for good optical access which limits model complexity, the need for in-situ calibrations, and degradation of the luminophore over a time-scale of hours[3]. Infra-Red Thermography is currently limited low resolution scans at frame-rate required for time-resolved measurements in high enthalpy wind tunnels[4]. Both methods can also suffer from measurement complications due to non-transparent and radiating flows.

Thin-film heat transfer gauges which are commonly used in low enthalpy testing only survive one shot when mounted on blunt body geometry in high enthalpy facilities[5, 6]. Surface junction thermocouples [7–10] are very robust but often require re-abrading or re-machining between shots due to damage from the flow. Additionally, the assumption of heat conduction into a homogeneous medium most often used to calculate heat flux from a temperature measurement[11] is not valid on a tens of micro-second time-scale[9, 12]. At longer time-scales reports of thermal product values for similar gauges vary by as much as 15 % [13, 14], and can vary by as much as 20 % depending on the method used to form the junction[12]. Good electrical grounding is required to avoid interference from ionised flows due to exposed

Table 1 Details of current types of DHTG. Estimated response times are calculated using Eq. (5).

DHTG Type	Diamond thickness, μm	Diamond Type	DHTG Numbers
1	150	Transparent PCD (TPCD)	1 to 5, 16, 17
2	220	Opaque PCD (OPCD)	6 to 10, 19
3	325	Transparent PCD (TPCD)	11 to 15, 18, 20

metallic junctions, which is difficult to achieve in practice.

This paper describes the development and testing of the Diamond Heat Transfer Gauge, a new thin-film calorimeter gauge, designed for use in high enthalpy short-duration wind tunnels. The short response-time required is achieved using a calorimeter made from synthetic CVD (Chemical Vapour Deposition) diamond which has an extremely high thermal diffusivity, mainly due to its exceptionally high thermal conductivity of $1000 \text{ Wm}^{-1}\text{K}^{-1}$ to $2600 \text{ Wm}^{-1}\text{K}^{-1}$ [15, 16]. A platinum thin-film RTD is used to measure the temperature of the calorimeter accurately and quickly.

The only known study in the literature where CVD diamond is a component of a heat transfer gauge was published by Follador[17], however the focus of the paper is on the growth of a diamond layer onto a thin-film gauge substrate and no experimental heat flux or temperature measurements are reported. Initial investigation of the present gauge using CVD diamond as the calorimeter[18] was undertaken at The University of Oxford by Clark[19] and Vanyai[20]. The present paper details the investigation of the concept, the development of the heat transfer gauge, and an experimental campaign in the X2 expansion tunnel at the University of Queensland where measured heat flux and robustness were compared to abraded surface-junction thermocouples[7].

II. The Diamond Heat Transfer Gauge

A cross-sectional schematic and photograph of the gauge are shown in Fig. 1. The calorimeter element of the DHTG consists of a disk of CVD diamond with a platinum thin-film Resistance Temperature Detector (RTD) sputtered on the rear surface. The diamond piece has a diameter of 2.52 mm and a thickness of 150 μm , 220 μm , or 325 μm , and is bonded into a steel housing using a low thermal conductivity adhesive with a high working temperature. The range of gauges detailed in Table 1 were made with different diamond thicknesses and diamond types to study the effect of calorimeter thickness on response time, accuracy, and durability. Gauges with 75 μm thick single crystal diamond calorimeters have been manufactured but were not used in this study.

A. Calculation of Surface Heat Flux

For a thin-skin calorimeter, surface heat flux can be calculated by measuring the transient temperature of the calorimeter, and subsequent analysis of the heat balance. Figure 2 shows a schematic of a two-layer semi-infinite system for which the equation for the diffusion of heat can be written as Eq. (1), where $n = 1$ is the calorimeter layer and $n = 2$

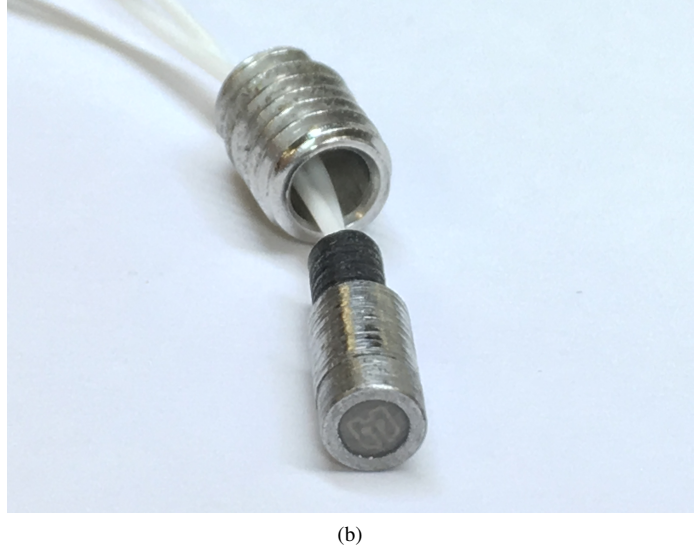
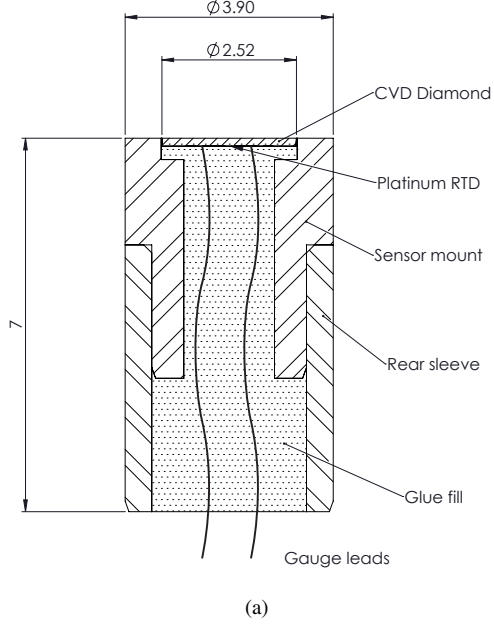


Fig. 1 A schematic (a) and photo (b) of DHTG20. The serpentine platinum RTD track is visible through the transparent PCD diamond in (b).

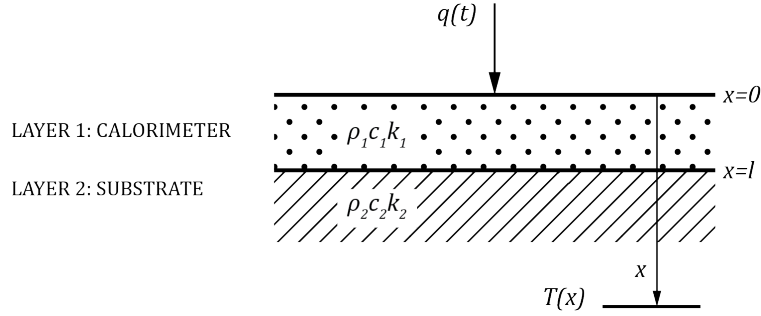


Fig. 2 Schematic of a one-dimensional analysis of a calorimeter gauge.

is the substrate.

$$\frac{\partial T_n}{\partial t} = \alpha_n \frac{\partial T_n}{\partial x^2} \quad (1)$$

The surface heat flux can be written using the calorimeter equation with temperature dependent thermal properties:

$$q_s(t) = \int_0^l \rho_1(T) c_{p,1}(T) \frac{dT(t)}{dt} dx + q_{loss}(t) \quad (2)$$

Defining the mean temperature as \bar{T} , Eq. (2) can be written:

$$q_s(t) = \rho_1(\bar{T}) c_{p,1}(\bar{T}) l \frac{d\bar{T}(t)}{dt} + q_{loss}(t) \quad (3)$$

Assuming radiative losses are negligible, the only loss from layer 1, the calorimeter, in the system shown in Fig. 2 is conduction into the substrate.

If the temperature at $x = l$ is known or measured (T_l), the loss can be calculated assuming one-dimensional conduction into a semi-infinite medium[11]:

$$q_{loss}(t) = \sqrt{\frac{\rho_2 c_{p,2} k_2}{\pi}} \left[\frac{T_l(t)}{\sqrt{t}} + \int_0^t \frac{T_l(t) - T_l(\tau)}{(t - \tau)^{3/2}} d\tau \right] \quad (4)$$

Equation 3 requires knowledge of the mean temperature of the calorimeter and the derivative of the mean temperature, with respect to time, to reproduce the surface heat flux. It is necessary to make the two assumptions relating the measured temperature to the mean temperature: firstly that $dT_l = d\bar{T}$ and secondly that $dT_l/dt = d\bar{T}/dt$. Section III presents a model of the DHTG where these assumptions are investigated.

B. Material Properties and Response Time

Material properties at room temperature of two grades of synthetic diamond are compared to Silicon Carbide (SiC) and copper (two alternative calorimeter materials) in Table 2. The variation with temperature of thermal conductivity and specific heat capacity for CVD diamond are shown in Fig. 3. The specific heat capacity is assumed to be the same for both grades of diamond[21]. The density of diamond has insignificant temperature dependence over the range of interest.

Schultz and Jones[11] give Eq. (5) as approximation for the response time of a calorimeter defined as the time taken for the rear surface temperature gradient to reach 99 % of the steady state value in response to a step change in heat flux at the front surface. Response time can be decreased by using a thinner calorimeter with a higher thermal diffusivity. The approximate response time of a calorimeter gauge using the materials in Table 2 is plotted in Fig. 4 as a function of thickness. For a given thickness of calorimeter, using transparent PCD instead of copper can give an order of magnitude reduction in response time. The approximate response times of the DHTG types detailed in Table 1 are 10.6 μ s for Type 1, 33.0 μ s for Type 2, and 49.6 μ s for Type 3.

$$\tau_r = \frac{l^2}{2\alpha_1} \quad (5)$$

C. RTD Calibration

During an experiment, as the temperature of the platinum film changes, so does its electrical resistance. The film is supplied with a constant current of approximately 10 mA, and change in resistance can be detected as a change in voltage. The experimental temperature change is calculated using Eq. (6), where V_{ref} is the voltage measured with the gauge at isothermal conditions at a temperature T_{ref} prior to an experiment. The coefficient of resistance with

Table 2 Thermal properties of possible calorimeter materials at 20 °C [15, 22, 23].

Material Property	Units	Transparent PCD (TPCD)	Opaque PCD (OPCD)	SiC	Copper
Density, ρ	kgm^{-3}	3514	3514	3100	8960
Specific Heat Capacity, c_p	$\text{Jkg}^{-1}\text{K}^{-1}$	495	495	750	385
Thermal Conductivity, k	$\text{Wm}^{-1}\text{K}^{-1}$	1852	1274	120	380
Thermal Diffusivity, α	$10^{-6} \text{ m}^2\text{s}^{-1}$	1064	732	52	114

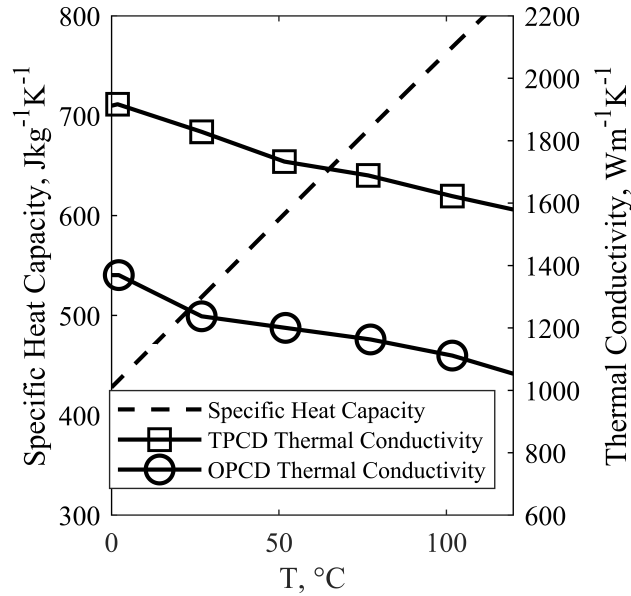


Fig. 3 Variation of thermal conductivity and specific heat capacity with temperature for transparent PCD diamond [15, 22].

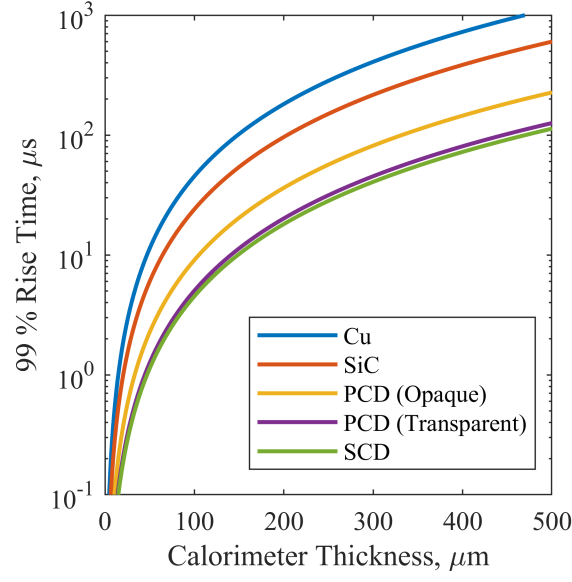


Fig. 4 The approximate rise time of an ideal calorimeter at room temperature as a function of thickness, Eq. (5). Plotted for materials shown in Table 2.

temperature (α_R) for the RTD in each DHTG was found using the calibration process described by Collins[24].

$$T(t) = \frac{V(t) - V_{ref}}{\alpha_R V_{ref}} + T_{ref} \quad (6)$$

III. Finite Element Heat Conduction Model

Although Schultz and Jones provide analytical equations for temperature at any depth in the simplified system shown in Fig. 2, the specific heat capacity and thermal conductivity of diamond are temperature dependent, as shown in the previous section. To assess the accuracy of the surface heat flux calculation with the assumptions presented in the previous section, a two-dimensional axisymmetric Finite Element model of a DHTG was used to model transient heat conduction in Comsol Multiphysics®[25].

Only the front 0.75 mm of Type 1 DHTG was modelled, as shown in Fig. 5. The 0.6 mm depth of adhesive is greater than the thermal penetration depth after 2000 μs . The mesh consisted of 15,500 triangular elements clustered towards the top surface and material boundaries. The two electrical gold leads of 60 μm diameter were modelled as a single lead of 84.85 μm , preserving cross-sectional area. The thermophysical properties for the diamond were taken from Ref.[15]; those for steel were taken to be $\rho = 7850 \text{ kgm}^{-3}$, $c_p = 475 \text{ Jkg}^{-1}\text{K}^{-1}$, $k = 44.5 \text{ Wm}^{-1}\text{K}^{-1}$; those for the epoxy were taken to be $\rho = 1500 \text{ kgm}^{-3}$, $c_p = 1000 \text{ Jkg}^{-1}\text{K}^{-1}$, $k = 0.55 \text{ Wm}^{-1}\text{K}^{-1}$; and those for gold were taken to be $\rho = 19300 \text{ kgm}^{-3}$, $c_p = 129 \text{ Jkg}^{-1}\text{K}^{-1}$, $k = 314 \text{ Wm}^{-1}\text{K}^{-1}$.

A constant heat flux of $q_{applied} = 10 \text{ MW/m}^2$ was used as the boundary condition across the top surface of the

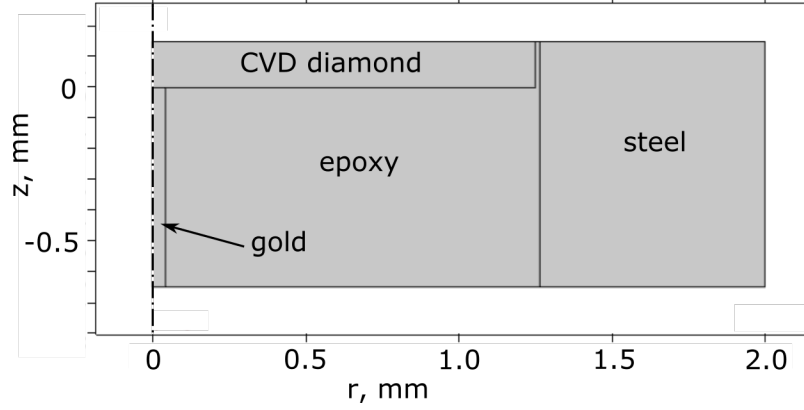


Fig. 5 The domain for the Finite Element model with material regions labelled.

domain. Adiabatic boundary conditions were used on all other external boundaries. The surface integral of temperature across the front (T_{front}) and rear (T_{rear}) side of the diamond were recorded, and are plotted in Fig. 6. The rear surface temperature output resembles that which the thin-film RTD would measure in the physical gauge. The difference in temperature across the diamond layer does not exceed 0.6 K in the simulation, therefore the assumption that $dT_l = d\bar{T}$ is valid, and spatial differences in material properties can be ignored when calculating surface heat flux.

The surface heat flux was calculated from T_{rear} using Eq. (3). The result is shown in Fig. 7a, normalised by the actual input to the model. Figure 7b shows the measurement error of the calculated heat flux. In both plots three lines are shown: the first (blue line) is calculated using Eq. (4) to correct for conductive losses from the diamond to the substrate, and with temperature dependent diamond properties; the second (orange line) is calculated in the same way but with constant diamond properties at the starting temperature; the third (yellow line) is calculated assuming no conductive losses from the calorimeter but temperature dependent diamond properties. By assuming that there are only losses through axial conduction, the heat flux calculated is accurate to within 3 % after the initial rise time. Losses through radial conduction into the glue and steel, and axial conduction through the gold electrical wires account for the remaining error. If the change in specific heat capacity with temperature is neglected, the error in measured heat flux is significant. The estimated response time of $14.8 \mu\text{s}$ from this study is higher than the $10.9 \mu\text{s}$ estimated by Eq. (5) due to conduction out of the diamond into the adhesive.

IV. Experimental Set-up

A. Facility and Conditions

The X2 Expansion Tube facility is a free-piston driven pulse flow wind tunnel which can produce scaled test flow conditions representative of atmospheric entry into most planets in our solar system[26]. X2 has multiple modes of operation, but for the purposes of this test campaign, it was operated as an expansion tunnel with a single driver tube and a diverging nozzle at the test section entrance. Details of X2 can be found in Gildfind et al.[27].

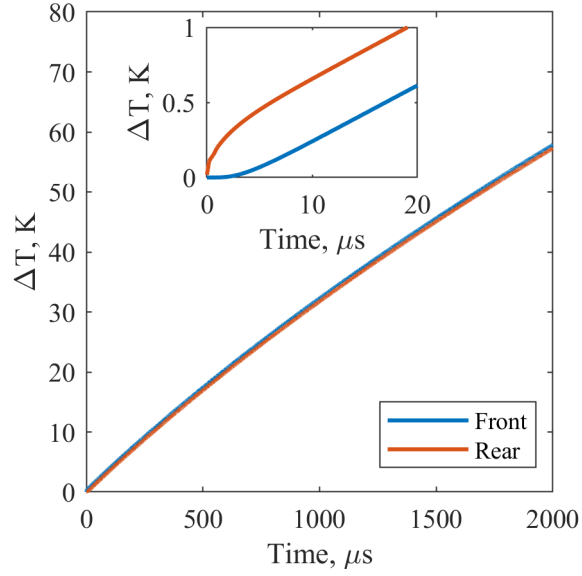


Fig. 6 Spatially averaged temperature rise across the front and rear surface of the diamond layer in the Finite Element model in response to a 10 MW/m^2 heat flux applied at the surface.

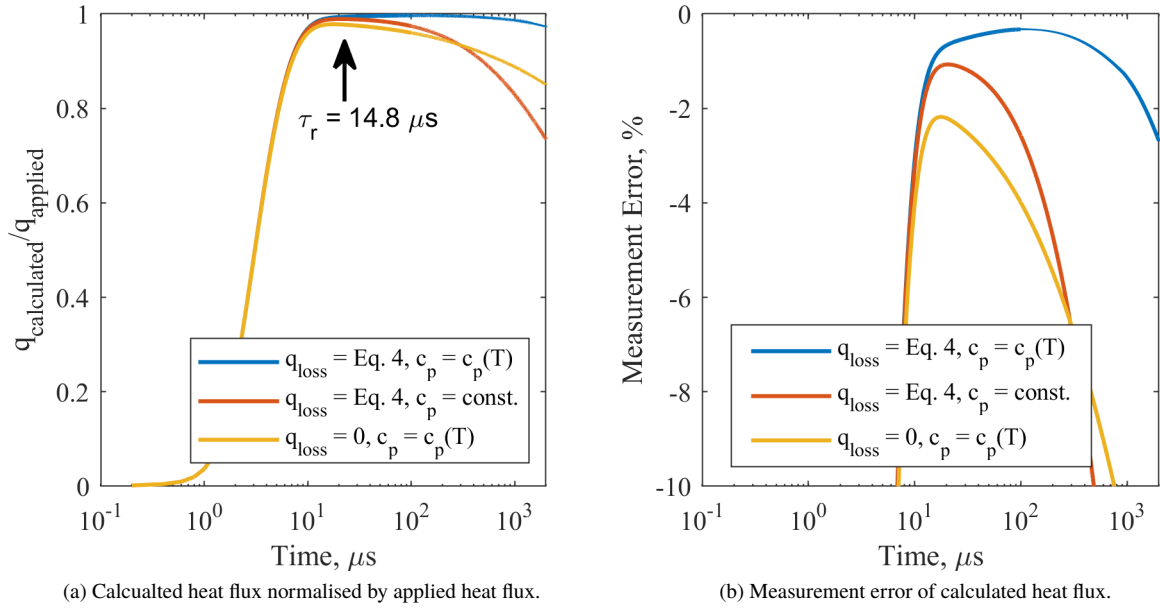


Fig. 7 Surface heat flux calculated using Eq. (3) from the temperature signal shown in Fig. 6 compared to the applied surface heat flux.

Table 3 Details of the conditions used in the X2 experimental campaign. All conditions used air as the test gas.

Condition	H_0 (MJ/kg)	U_∞ (km/s)	ρ_∞ (g/m ³)	p_∞ (kPa)	T_∞ (K)	M_∞	Leading Edge	Shot Numbers
A [30]	25.3	7.0	1.7	0.62	1300	9.9	Sharp Blunt	x2s3482 - x2s3484 x2s3504 - x2s3506
B [31]	39.6	8.7	1.0	0.58	2000	10.3	Sharp	x2s3490 - x2s3491
C [32]	53.4	10.1	2.0	1.07	1900	12.2	Sharp	x2s3492 - x2s3494
D [33]	70.7	11.4	1.9	1.99	3150	10.3	Sharp	x2s3495 - x2s3498
E [34]	50.0	8.3	15.3	35.04	5850	5.2	Blunt	x2s3507 - x2s3510

The nominal test section flow properties for each condition used are shown in Table 3. The flow conditions were calculated from facility fill-pressures and measured shock speeds using PITOT, a University of Queensland in-house facility simulation code[29]. This set of flow conditions allowed the robustness and response of DHTGs to be evaluated across a range of flow enthalpies and surface heat fluxes.

B. Model and Instrumentation

The test model used was a 1/33 scale model of a simplified centreline profile of the IXV entry vehicle[35]. The profile is simplified to a blunt-nosed wedge, with a nose radius of 21.7 mm and the overall length of 140 mm. In this configuration the angle of attack was set to 45°. The blunt nose is replaceable with a sharp leading edge, transforming the model into a 22.5° wedge. In this configuration the top of the model was set to 0° angle of attack. Figure 8 shows both configurations of the model with hole numbers for heat transfer gauges and a PCB transducer sensor labelled.

The model was instrumented with a total of 10 DHTGs, 8 coaxial surface-abraded thermocouples[12], 2 thin-film heat transfer gauges, and 1 PCB 113B pressure transducer. The top side had five DHTGs (holes 1 to 5), three thermocouples (holes 6, 8, and 10), and two thin-film gauges (holes 7 and 9). The underside had five DHTGs (holes 11 to 15) and five thermocouples (holes 16 to 20). Two additional DHTGs were mounted on the stagnation point when the blunt leading edge was attached (holes 21 and 22). This arrangement of heat transfer gauges allowed comparison between the different types of gauge at equal distances from the leading edge. During the experimental campaign DHTGs which were significantly damaged or broken were replaced. The thermocouples, which were all made to the same specification, were glued permanently into the model and were not moved or replaced during the course of the test campaign. Constant current supply and signal amplification for the DHTGs was provided by the University of Queensland HTG3 amplifiers which have a bandwidth of 350 kHz. The thermocouple amplifier used had a response time of 1 μ s.

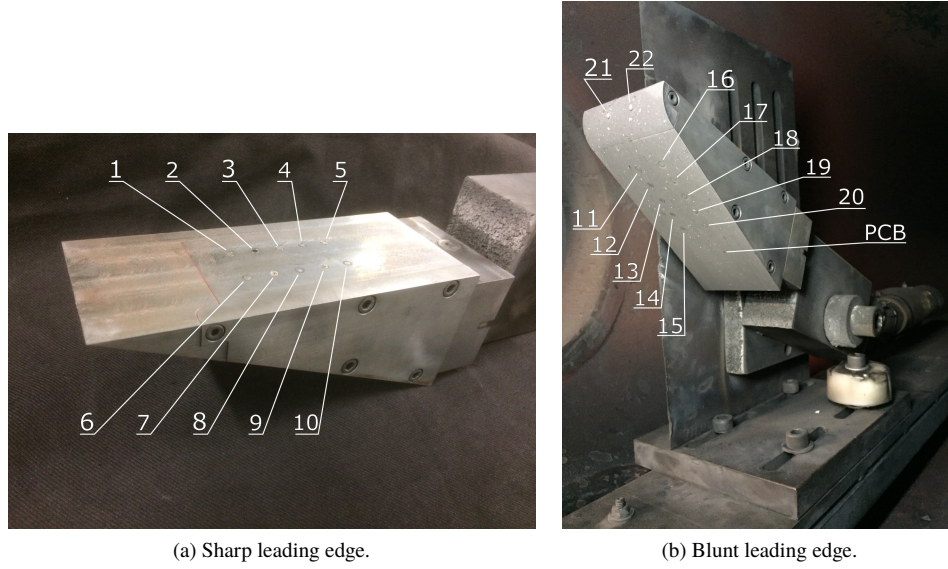


Fig. 8 Photos of the model used for experimental testing in X2 with hole numbers labelled.

C. Data Processing and Measurement Uncertainty

Experimentally recorded voltages were first processed by removing the gain of the amplifiers. For the DHTGs the de-boosted voltage signal was processed into temperature using Eq. (6), and subsequently into heat flux using Eq. (3). The specific heat capacity and density of the diamond used in the equation were updated at each time-step according to the measured thin-film temperature and the temperature-dependent properties presented in Fig 3. For the thermocouples, temperature was calculated using a sensitivity of $62.5 \mu\text{V K}^{-1}$ [13]. To calculate the heat flux, the impulse response method developed by Oldfield[36] was used, using the standard assumption of one-dimensional semi-infinite conduction into a homogenous medium. The effective value of thermal product ($\sqrt{\rho c k}$) was taken to be $8900 \text{ Js}^{1/2}\text{m}^{-2}\text{K}^{-1}$ [13]. The heat flux signals from the DHTGs and thermocouples were filtered using a low-pass filter with a cut-off frequency of 100 kHz and 500 kHz respectively.

The uncertainty in the heat flux measured by the DHTGs was calculated to be 11.1 % by combining the uncertainty associated with each element of Eq. (3) using the the Root-Sum-Square method[37]. The greatest contributing factor is the uncertainty in the specific heat capacity of diamond, estimated to be 10 %. The uncertainty in the heat flux measured by the thermocouples is dominated by uncertainty in the value of effective thermal product used: the real value depends on the exact location of the thermocouple junction, how the junction was made, and the time-scale of interest. Estimates of measurement uncertainty on a time-scale of milliseconds for similar gauges are at least 10 %[12, 14]. On a time-scale of tens of microseconds, which is of interest for the current study, the thin layer of electrical insulation between the two metals can affect heat conduction through the gauge and decrease the effective thermal product by as much as 30 %[9], leading to an over-prediction in surface heat flux measurement.

V. Experimental Results

A. Surface Heat Flux

The surface heat flux measured by a DHTG and a thermocouple for one shot at each flow condition are compared in Fig. 9. The heat flux traces all show a characteristic peak upon first arrival of accelerator gas, followed by the arrival of the test gas and a period of relatively steady heat transfer which is the useful test time. The end of the test time is dictated by contamination of the test flow with driver gas, indicated by a significant rise in heat transfer. The heat fluxes measured by both gauge types agree well, with the exception of Condition A and Condition E, where the DHTG measures a significantly lower heat flux. The initial peak measured by the DHTG is also smaller due to the slower response time of the DHTG.

Heat flux measurements from all three shots at Condition E, the highest enthalpy condition, are presented in Fig. 10. Each row of plots represents a different shot, while each column represents a different axial location from the leading edge, with the first column being the stagnation point. In each plot the measured heat flux from a DHTG and a thermocouple at equal distances from the leading edge are compared. On the stagnation point the heat flux from a DHTG compared to an empirical correlation developed by Sutton and Graves for generic axisymmetric blunt bodies with gases in chemical equilibrium[38]. For the non-stagnation point measurements, the thermocouples read a higher and more oscillatory heat flux than the DHTGs, but the agreement is generally good.

On some shots in the experimental campaign, the measured heat flux from both types of heat transfer gauges have large oscillations. This is most noticeable for DHTGs at low enthalpy conditions where the signal-to-noise ratio was poor and for the thermocouples at higher enthalpy conditions where the post-shock gas temperature was sufficiently high to produce ionisation which may interfere with the temperature measurement. The heat flux signal from both types of gauge was filtered using a low-pass filter with a cut-off frequency of 200 kHz. Based on the response times shown in Table 1 and assuming a first-order system, this filter does not remove any of the useful signal content from the DHTG signal, i.e. content produced by surface heat flux oscillations. Although the quoted response time of the thermocouples is of the order of microseconds, a filter that does not attenuate high frequency components (greater than 200 kHz) of the signal leaves oscillations on the order of 100 MW/m^2 in the heat flux trace which are assumed to be non-physical.

B. Robustness

Between each shot in the experimental campaign, a photo was taken of the surface of each heat transfer gauge and the electrical resistance across the thin-film in each DHTG was recorded with a multimeter. The photos of the gauges in Fig. 11 show evidence of the extreme environment to which the gauges were exposed. Craters produced by impacts from diaphragm fragments can be seen on the surface of the steel model in all photos with the exception of Fig. 11a, which was taken prior to any testing.

The survival rate for DHTGs on the flat plate side of the model was 100 %. Of the ten DHTGs that were mounted at

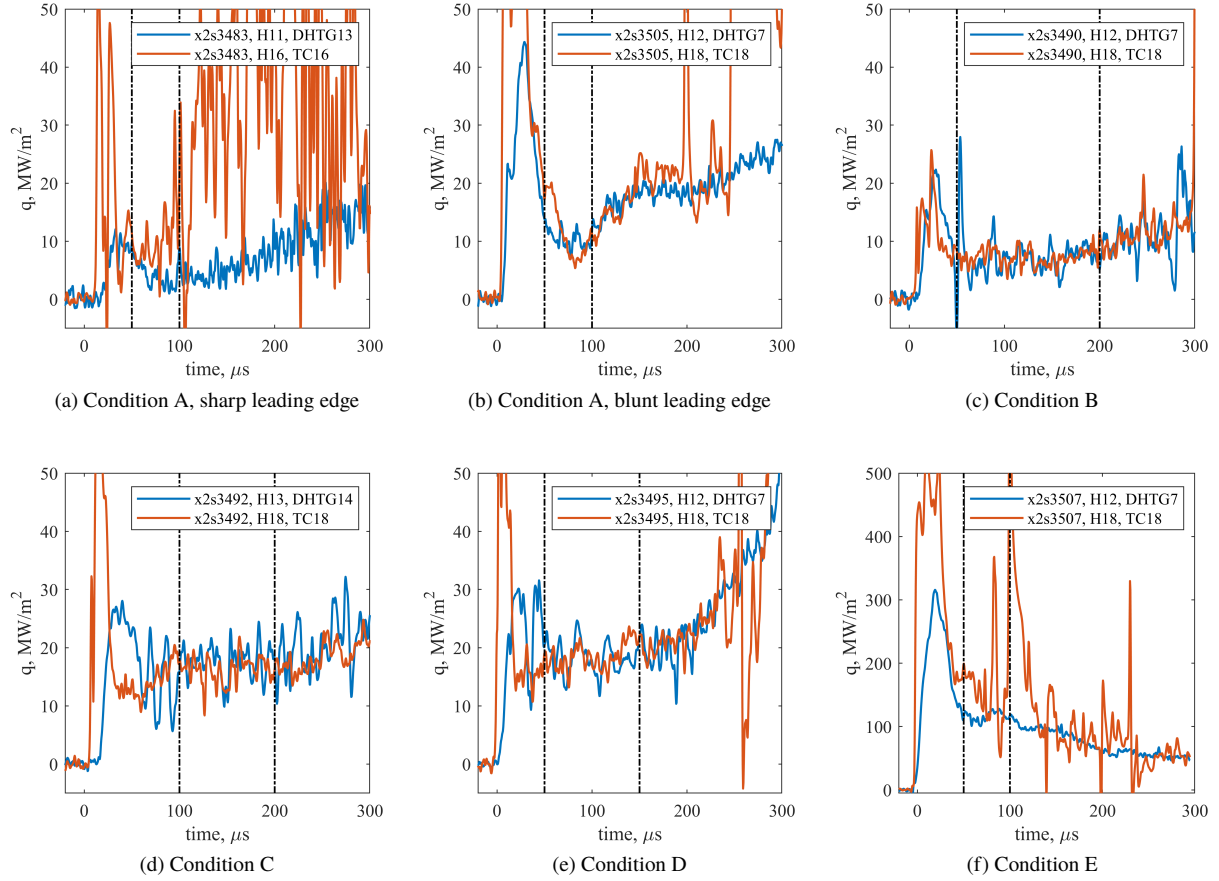


Fig. 9 Surface heat flux measured by a DHTG and a thermocouple for one shot at each condition in the experimental campaign. The test time lies between the vertical dashed lines.

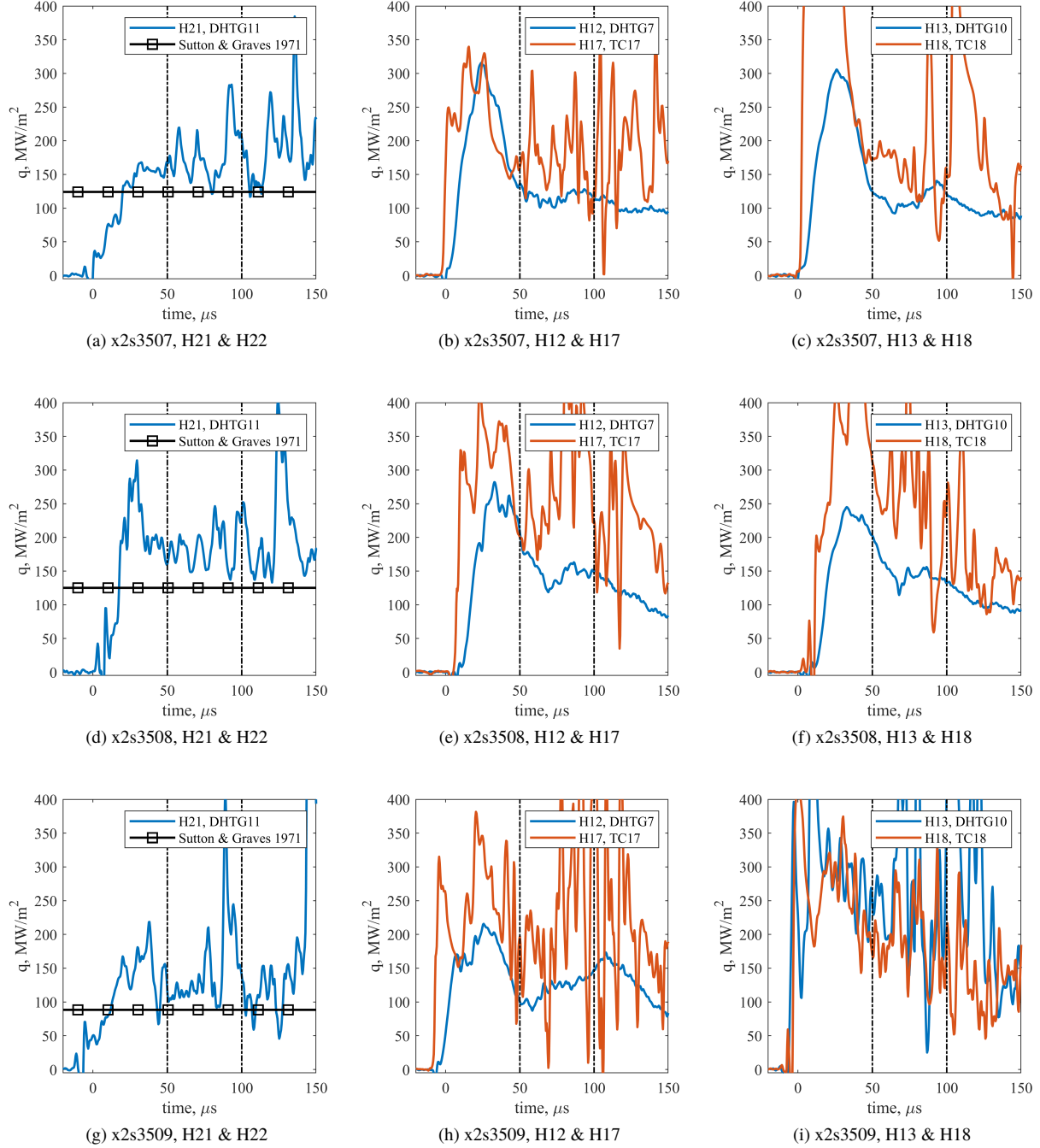


Fig. 10 Measured heat flux from DHTGs and thermocouples for Condition E. Each plot shows gauges the same distance from the leading edge. The useful test time lies between the vertical dashed lines.

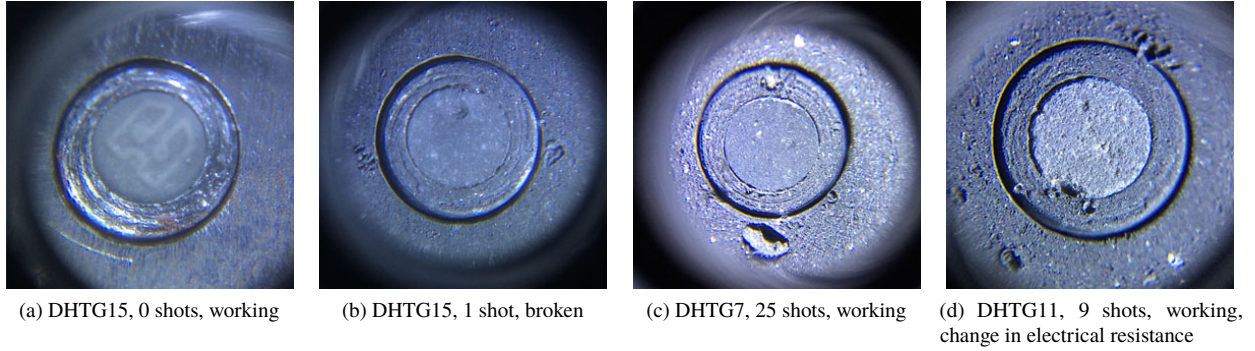


Fig. 11 Photos of the surface of DHTGs before, during, and after experimental testing.

an angle of attack, four ended the campaign without any sign of damage having been used for 17 shots on average. Four gauges were destroyed, surviving 1.75 shots on average. Two gauges showed a change in resistance across the thin-film but were still functional. A re-calibration of gauges after the campaign had ended (see Section II.C) showed that a change in room-temperature resistance of more than 1Ω indicates significant deviation from the original calibration, and that therefore the heat flux measured is unreliable. Photographs of the surface of the DHTGs suggest that a singular impact from a large diaphragm fragment can break the thin-film on the rear surface of the diamond. Many smaller impacts have the effect of surface abrasion and damage does not propagate to the rear surface and damage the thin-film. The sample size was not large enough to draw conclusions about the effect of diamond thickness on robustness. The thermocouples required abrading between shots to re-make the surface junctions, however all but one of the thermocouples used was still functional at the end of the campaign.

VI. Conclusion

A new fast-response calorimeter heat transfer gauge has been designed, fabricated, and experimentally tested. The DHTG is shown to be a viable alternative to surface abraded coaxial thermocouples for measuring surface heat fluxes in high enthalpy pulse flow wind tunnels. Although The DHTG is more susceptible to damage from diaphragm particles than thermocouples at an angle of attack in an expansion tube environment, the robustness demonstrated is promising for planned future testing in reflected shock tunnels, which have less damaging flow conditions at the cost of total enthalpy.

The measurement uncertainty of the DHTG has been quantified and is comparable to that of coaxial thermocouples. Calibration experiments involving direct measurement of the impulse response of DHTGs are planned, the results of which will reduce the heat flux measurement uncertainty by removing the need to assume material properties and physical dimensions of the gauge.

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