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Date of resubmission: 06/11/2024

Manuscript version – After 2nd review

Title: Comparison of 1g and centrifuge modelling of drag embedment anchors with subsurface wireless tracking

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This is the accepted version of: Sharif, Y, Brown, M, Knappett, J, Davidson, C, Bird, RE, Coombs, WM, Augarde, C, Carter, G, Macdonald, C & Johnson, K 2024, 'Comparison of 1g and centrifuge modelling of drag embedment anchors with subsurface wireless tracking', International Journal of Physical Modelling in Geotechnics. <https://doi.org/10.1680/jphmg.24.00029>

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Number of words : 7193

Number of figures: 9

Number of Tables: 3

Notations

H	Penetration depth
a_{k-1}	acceleration vector
A_x	Acceleration in x direction
A_y	Acceleration in y direction
A_z	Acceleration in z direction
D_{10}	effective particle diameter
D_{50}	average particle diameter
D_r	relative
N	Scaling factor
	measurement noise covariance
R	matrix
v_k	velocity vector
X_k	current state of the anchor
x_k	current position in x direction
y_k	current position in y direction
z_k	measurement model
β	Anchor opening angle
δ'_{crit}	sand-steel interface friction angle
Δt	timestep
θ	angle in a given direction
θ_a	angle reported from accelerometer
μ_a	gyroscope data
ρ_{max}	maximum dry density
ρ_{min}	minimum dry density
ϕ'_{crit}	critical state friction angle
ϕ'_{pk}	peak friction angle
ψ	dilation angle
S_x	standard deviation of the position
$S_{\dot{x}}$	standard deviation of the velocity
\dot{x}_k	velocity in x direction
\ddot{x}_k	Acceleration in the x direction

Abbreviations

CBRA	Cable burial risk assessment
DEA	Drag embedment Anchor Integrated development
IDE	environment
IMU	Inertial measurement unit
MEMs	micro-electromechanical
NI	National instruments
NiCAD	nickel cadmium
SF	Seabed factor Technical data management
TDMS	streaming
UHC	Ultimate holding capacity

1 Abstract

2 The kinematic behaviour of drag embedment anchors has become a recent research focus
3 due to the increase in offshore renewable energy devices. This is due to their potential use
4 as an anchoring system for future floating wind applications, in addition to the need to
5 understand their penetration behaviour as a part of the cable burial risk assessment (CBRA).
6 Studies on the behaviour of anchors typically consist of field scale or model centrifuge tests,
7 where such facilities are not readily available to all and can result in significant cost. In
8 addition to this, measuring the load-penetration behaviour of an anchor has proven to be a
9 significant challenge, as any contact-based methods are likely to influence the penetration
10 behaviour of the anchor. In this paper a novel wireless method of recording the inclination
11 of the anchor and calculating the penetration depth is presented. A comparison of the
12 penetration behaviour of a Class F (AC-14) anchor has been investigated in sand using
13 centrifuge and 1-g model scale testing. The results indicate that the 1-g testing can match
14 the behaviour of the anchor testing in the centrifuge in terms of both the position of the
15 anchor and its orientation during the dragging event.

16 Keywords: Drag embedment anchors, centrifuge modelling, instrumentation

17 2 Introduction

18 Drag embedment anchors (*DEAs*) are starting to attract a new more recent research focus
19 due to offshore renewable energy applications. This may be as a potential anchoring system
20 for future floating wind applications (Cerfontaine *et al.*, 2023; Davidson *et al.*, 2023) or due
21 to a need to understand their behaviour as a part of a Cable Burial Risk Assessment
22 Framework (*CBRA*) (Carbon Trust, 2015; Sharif *et al.*, 2023). To be able to inform such

23 investigations, it is necessary to understand not only the load displacement of the anchor but
24 also how it transitions from the seabed surface to a final steady state if pulled to its ultimate
25 holding capacity (*UHC*). This includes how the anchor fluke pitches (Figure 1b) and how
26 the depth evolves to the anchor's final position and orientation. *CBRA* methodologies often
27 attempt to predict the anchors final penetration depth based upon a simplistic soil dependent
28 seabed factor multiplied by the fluke length and the angle of the opening of the fluke relative
29 to the shank. (Equation 1) (Carbon Trust, 2015).

$$H = SF \times \sin \beta \times F \quad (1)$$

30 where H is the penetration depth of the anchor, SF is the seabed factor (1 for “Hard soils”
31 and 4 for “Soft soils”), F is the fluke length of the anchor and β is the opening angle of the
32 anchor. The assumption of Equation 1 is that the shank is horizontal (Figure 1) to the seabed
33 and that the opening angle of the anchor is the maximum possible for that anchor type. From
34 a previous centrifuge study by Sharif et al (2023) it was found that both assumptions may
35 not always be correct as the values change with soil density and there is a lack of available
36 data regarding the actual opening angle in reality due to the difficulty in measuring this
37 property once the anchor has been deployed.

38 Information on the behaviour of DEAs is difficult to obtain during the pull event itself in 1g
39 or centrifuge physical modelling experiments, as the anchor is likely to penetrate to some
40 depth below the soil surface, making observations of the continuous location and orientation
41 of the anchor during the anchor pull difficult. In previous studies the anchor depth has been
42 monitored by placing thin shafts on the fluke, that protrude above the soil surface. The length
43 and angle of the shafts were then monitored and used to determine the penetration depth of
44 the anchor (Neubecker and Randolph, 1996). However, the effect of the attached shaft could
45 result in the kinematic behaviour of the anchor changing. Thus, providing a penetration
46 depth or orientation that may not be accurate for the geometry of the anchor. Other methods

47 which have been employed are excavation of the soil around the anchor at its final location
48 (Moore *et al.*, 2021) and stopping the anchor drag at regular intervals to probe the soil in an
49 attempt to locate the anchor and its penetration depth. Excavation of the anchor at the end
50 of the drag distance, by first saturating the soil and then draining the water as was conducted
51 by Moore *et al.* (2021), which gives an accurate indication of the orientation and depth of
52 the anchor but does not provide any information on the transitional behaviour of the anchor.
53 Probing the soil during the dragging may give an indication of depth but true positioning
54 information is difficult to obtain as it is not always clear what part of the anchor, you're in
55 contact with and does not allow detailed information on fluke opening angles to be obtained.

56 Other attempts include the use of on-board 3 axis micro-electromechanical (*MEMs*)
57 accelerometers as used by Robinson *et al.* (2019) to track the progress of offshore pipeline
58 and cable ploughs, but these require wired connections that can be dragged behind the
59 anchor or transferred up the shank and towline (Davidson *et al.*, 2023). Similar to the
60 protruding shaft, the addition of a trailing wire has the potential to affect the kinematics of
61 the anchor or even damage the wiring or the *MEMs* chip to wire connection during testing.

62 In addition to the challenge of instrumentation, tracking and data transfer, anchors are often
63 complicated bespoke shapes that vary from manufacturer to manufacturer and do not lend
64 themselves to accurate reproduction at small scale for physical modelling. It is also the case
65 that not all researchers are fortunate enough to have access to a geotechnical centrifuge
66 facility and it is thus useful to determine which modelling activities are sensitive to stress-
67 based scaling when testing small models during large strain drag events. For instance,
68 Bransby *et al.* (2005) and Robinson *et al.* (2019) showed that with proper consideration large
69 deformation events such as pipeline ploughing could be adequately represented by 1g testing
70 without changing the size of the model plough.

71 This paper describes a comparison of both 1g and centrifuge testing of a common drag
 72 embedment anchor type fabricated using 3D metal printing in dry sand (assuming fully
 73 drained conditions). To allow comparison of kinematics of the model anchors during
 74 installation, a miniature wireless tracking system with real time monitoring was developed
 75 and is also described herein.

76 3 Methodology

77 The following section outlines the methodology and equipment used to model the scaled
 78 centrifuge and 1g experiments used within this study. The equipment used for the 1g and
 79 centrifuge tests were identical, with the only difference being the application of an enhanced
 80 gravitational acceleration in the centrifuge experiments.

81 3.1 Sand used and preparation

82 The sand beds used to test the anchor penetration, were dry pluviated into the large
 83 deformation strong box to a depth of 400 mm. Sand beds were created at four homogenous
 84 relative densities to assess how density affects the penetration of the AC-14 anchor as part
 85 of a wider CBRA study. For each of the chosen relative densities a 1g and a centrifuge box
 86 was prepared so that a comparison of the kinematic behaviour at each relative density could
 87 be made. The relative densities (D_r) chosen were 25%, 38%, 55% and 82% to represent sand
 88 bed in the loose, medium and very dense categories. The sand used in the experiments was
 89 HST95 sand which is a fine-grained quartz laboratory sand commonly used in the
 90 geotechnical laboratories of the University of Dundee.

91 *Table 1: HST95 sand material properties (Al-Defae et al, 2013; Lauder et al., 2013)*

Property	Value
Effective particle size D_{10} mm	0.09
Average particle size, D_{50} : mm	0.14

	Peak friction angle, ϕ'_{pk} , at 57% relative density: degrees	40
	Peak friction angle, ϕ'_{pk} , at 84% relative density: degrees	45
	Critical state friction angle, ϕ'_{crit} : degrees	32
	Sand–steel interface friction angle, δ'_{crit} : degrees	24
	Angle of dilation*, ψ : degrees	16
	Maximum dry density, ρ_{max} : kN/m ³	17.58
	Minimum dry density, ρ_{min} : kN/m ³	14.59
92	<hr/> Peak friction angle determined at effective stresses relevant to model testing (0.2–0.3 kN/m ²)	
93	Soil beds were created with air pluviation from an automated hopper fixed at a constant	
94	height. The pluviator was a slot type pluviator that extended past the edge of the strong box	
95	with the relative density controlled through changing the width of the slot at the lower edge.	
96	The same system was used for both the centrifuge and 1g sand beds.	

97 3.2 Centrifuge modelling

98 To replicate the prototype stress conditions, geotechnical centrifuge testing of a 1/24th scaled
99 model of an 8.5 tonne AC-14 anchor was conducted in dry sand at 16.4g. An AC-14 anchor
100 was chosen for this study, due to Luger (2023) suggesting that it is the most widely deployed
101 anchor in the shipping industry, in addition to having previously been investigated through
102 both physical model scale (Moore *et al.*, 2021), numerical modelling (Grabe *et al.*, 2015)
103 and field scale (Luger and Harkes, 2013) experiments. The g level the test was conducted at
104 was lower than the scaling factor in order to recreate the effective stress of a fully saturated
105 soil bed without the need for pore fluid or waterproofing of the instrumentation located
106 within the model anchor. An additional benefit to utilising dry sand beds for testing, is the
107 ease of formation of sand bed via air pluviation and the potential for quick test turn around,
108 as the sand does not require drying between experiments. This reduced g level approach in
109 dry sand to mimic drained saturated conditions was proposed by Li *et al.* (2010) and
110 validated by Klinkvort and Hededal (2013). This method utilises the ratio of the dry unit
111 weight and the buoyant unit weight of the soil and uses a lower g-level with dry sand to
112 produce the same stress profile as that of a saturated soil bed spun at the g level the scaling

113 factor is determined at. Therefore, a dry sand bed at 16.4g would represent a drained test in
 114 a saturated soil bed spun at 24g. Although this may not be fully indicative of field
 115 deployment of anchors, it would represent the worst-case scenario for an anchor penetration,
 116 as was shown by Grabe et al. (2015), who showed reductions in anchor penetration when
 117 the anchor is pulled under undrained conditions in sands of varying relative density, ranging
 118 from loose to dense. Further work is planned by the authors to characterise the anchor
 119 behaviour under saturated partially drained and undrained conditions, as this requires further
 120 development of the internal anchor instrumentation.

121 The model anchor was 3D metal printed from 316L stainless steel (Figure 2) ensuring that
 122 the mass and the centre of gravity of the anchor were scaled accurately, such that the
 123 behaviour of the anchor replicated that of the real anchor. Considerations in the design of
 124 the anchor were made to account for the mass of the instrumentation and batteries required
 125 to power them. The model and prototype scale dimensions of the anchor can be seen in Table
 126 2.

127 *Table 2: Properties of the model and prototype AC14 anchor geometry*

Property	Model scale	Prototype scale
Shank length (mm)	137.8	3308.0
Fluke length (mm)	71.46	1715.0
Fluke Width (mm)	90.25	2166.0
Mass (kg)	0.07	8700

128

129 The enhanced g experiments were conducted in the University of Dundee's 3.0 m radius
 130 beam centrifuge in a large horizontal deformation strong box. The strong box had internal
 131 dimensions of 1400 mm x 400 mm x 650 mm using a dedicated large displacement actuator,
 132 developed to investigate the performance of ploughs and anchors (Robinson *et al.*, 2017,

133 2019; Davidson *et al.*, 2023). The tow force was measured using a 5kN S-type loadcell
134 (Tedea Huntleigh type 616) positioned at the surface of the sand bed, which was attached to
135 the towing arm mounted to a moving platform. The moving platform was attached to the
136 motor by two timing belts looped around pulleys positioned at the ends of the actuator. The
137 pulleys were in turn connected to an axle driven by a secondary belt attached to a Paralux
138 SD12-LWS high torque 220V DC motor (capacity of 63 Nm at 13 rev/min). The
139 displacement of the platform was measured by a draw wire transducer (*DWT*) (Multi-comp
140 SP1-50) (Figure 3).

141 The anchor pad-eye was connected to the loadcell by a forerunner (tow cable) consisting of
142 a 1.5 mm diameter steel wire rope of length 420 mm (Figure 3a). A cable was selected in
143 place of a chain due to the difficulty in obtaining a chain scaled correctly that would be
144 strong enough to withstand the forces experienced in the centrifuge experiments, it was also
145 thought that a cable would represent a worst-case scenario due to its lower frictional
146 resistance and reduced flexibility. Thus, resulting in the anchor that penetrate deeper than
147 one installed using a chain. To determine the influence of a chain compared to a cable would
148 require a further study, out with the scope of the present study. A swivel and shackle were
149 located at the loadcell end of the forerunner cable to minimize torsional forces from
150 tensioning of the twisted wire rope. A 200g 3 axis accelerometer (Analogue Devices
151 ADXL377) was mounted to the swivel to measure the inclination of the forerunner during
152 the centrifuge tests, and a 3g 3 axis accelerometer was used for the 1g tests, which was used
153 as an indication of the shank angle of the anchor. The measurements of the accelerometer
154 were verified pre and post testing by taking manual measurements of the taught forerunner
155 cable using a digital clinometer. The values obtained from the digital clinometer were on
156 average $\pm 0.1^\circ$ different to that of the accelerometer due to the level of precision from the
157 clinometer. Based on excavations conducted of the anchor post testing, it was found that the

158 shank of the anchor was in-line with the forerunner cable in all densities tested, as such it
159 was assumed that the angle of the forerunner cable could be used as an indication of the
160 angle of the anchors shank. This assumption would only apply for the experiments and
161 would not be applicable for anchor deployment in the field, as it is unlikely that the
162 inclination of the towline/chain would be the same as that of the anchor shank in long chain
163 field situations. This is due to potential that a reverse catenary is formed in the soil by the
164 chain.

165 To control the actuator and record the signals from the analogue sensors a National
166 Instruments (NI) based control and data acquisition system was used with a purpose built
167 Labview 2018 virtual instrument (VI). The system is based around a CompactRio cRIO-
168 9024 in hybrid mode, utilising both the Scan engine and field programmable gate arrays
169 (FPGA) for analogue output (instrumentation power and relay switching, where the relays
170 control the power state and direction of the motor) and analogue input (reading signal data
171 from instrumentation). To provide the required stable voltage to the wired analogue sensors
172 a NI-9264 C-series module was utilised and a NI-9202 C-series modules was used to
173 simultaneously read the analogue channels. A logging rate of 50Hz was used for all
174 instrumentation, as this was deemed adequate for the relative slow drag velocity (20
175 mm/min) of the model anchor. The drag speed was chosen to maximize the amount of data
176 that was recorded by the logging system in the relatively short time available from the
177 batteries used in the anchor internal instrumentation. As the test was conducted under
178 drained conditions, this did not influence the results of the experiment.

179 Testing started with the anchor on the sand surface, such that the opening angle was 0° and
180 test progressed by advancing the platform away from the anchor at a rate of 20 mm/min.
181 The test was typically stopped after 1000 mm of displacement. Once the test had been
182 conducted, and the centrifuge had spun down, the soil surrounding the anchor was partially

183 saturated using water and the anchor was carefully excavated to take measurements of the
184 anchors final penetration depth, shank angle and fluke angle to confirm the readings
185 recorded by the onboard logging system during the test. The anchor attitude (orientation
186 relative to the seabed) was measured using a digital inclinometer and the anchor depth
187 determined relative to the sand bed surface using a vernier calliper.

188 3.3 1g testing

189 To undertake the 1g testing the same equipment and testing procedure as described for the
190 centrifuge experiments was used (Figure 3b). The only difference between the two sets of
191 experiments is that the centrifuge-based experiments were conducted at 16.4g whereas the
192 1g experiments were conducted on the lab floor without the use of the centrifuge. This
193 allows for a direct comparison of the experiments.

194 3.4 Instrumentation development and operation

195 The towline accelerometer was monitored with the *NI* based logging system through a wired
196 connection that travelled up the towing arm. The internal system inside of the anchor was
197 based around a low cost Seeed Studio XIAO nRF52840 Sense Bluetooth development board
198 which is an Arduino compatible board (Figure 2b). The development board features an on-
199 board Bluetooth antenna (for transmitting and receiving data) and a 6 axis inertial
200 measurement unit (*IMU*) containing a 16g 3 axis accelerometer and a 3 axis gyroscope. Due
201 to the low measuring range of the digital accelerometer built into the development board, a
202 secondary 200g accelerometer was wired into the Seeed Studio board for transmitting to the
203 logging PC. To program the Seeed Studio XIAO nRF52840 Sense a variety of programming
204 languages and software packages can be used, including but not limited to Arduino IDE,
205 MicroPython, CircuitPython, in the case of this study Arduino Integrated Development
206 Environment (*IDE*) was utilised with C++ being the chosen programming language.

207 The onboard anchor Bluetooth instrumentation was continuously logged at a rate of 50Hz.
208 To process the data a Kalman filter was used to calculate the state of the anchors position
209 and orientation based on the measured 6 axis *IMU* data. Since undertaking the experiments
210 outlined in this study, the logging system for the wireless sensors has been updated so record
211 the data concurrently with that of the wired sensors. The Bluetooth signal for the updated
212 system was received by a non-sense XIAO BLE development board from the XIAO
213 nRF52840 Sense in the anchor and the data written through a serial connection to the
214 CompactRIO, such that the wireless instrumentation and the wired instrumentation are
215 logged concurrently. The data is then logged directly in Labview to a Technical data
216 management streaming (*TDMS*) file.

217 The XIAO nRF52840 used in this study was programmed to broadcast the data to the
218 logging PC. The transmitting unit was powered by two nickel cadmium (NiCad) coin-cell
219 batteries to provide the required voltage to the development board. NiCad batteries were
220 used in place of the more power dense Lithium based battery types (lithium ion and Lithium
221 polymer) due to safety concerns. Lithium battery technology is considered too volatile for
222 the forces experienced in centrifuge testing, due to the potential damage that may occur
223 when spinning up and as such are not used in the University of Dundee geotechnical
224 centrifuge. As a result of using the NiCad batteries the available transmission time of the
225 development board is significantly reduced. Such that for a single battery pack only 30 mins
226 of power is available. Therefore, the battery pack is changed for each test.

227 Instrumentation embedded on the XIAO BLE Sense board and attached to the anchor
228 through wires were powered through the voltage regulator chip on the board. To extend the
229 available charge on the coin-cell batteries, the development board was initially placed in
230 standby mode, in which none of the instrumentation is powered and no transmission of data
231 is occurring. Once the centrifuge had reached the desired g-level, a signal was sent from the

232 logging PC to “wake” the development board up, at which point the data is read from the
233 onboard gyroscope and accelerometer and concatenated with the timestamp of the reading
234 and then transmitted to the logging PC. For each timestep or cycle of data reading, the
235 concatenated string is transmitted and received, such that the logging PC received the data
236 at a frequency of 50Hz. Before commencing the test, the logging PC received 10 seconds of
237 data to establish a baseline and to ensure that the data was being transmitted effectively.
238 Once it had been established that the signal is stable, the test commences. At the end of the
239 drag event, the data is recorded for an additional 30 seconds before a sleep command is
240 transmitted to the development board in the anchor to place it back into standby mode.

241 The data for the wired instrumentation (towline accelerometer, loadcell and *DWT*) were
242 recorded from the moment the centrifuge starts to accelerate, and recording ends when the
243 centrifuge has decelerated and has come to a standstill. To synchronize the data from the
244 wired and wireless sensors, inclination of the towline accelerometer and the point at which
245 the loadcell records a force are assessed. As the forerunner cable is initially slack, the
246 inclination of the accelerometer on the swivel reads a value that is high in pitch direction
247 (Figure 1b). When the sled moves the forerunner becomes taut and as the loadcell is
248 positioned at the surface of the soil, the pitch of the accelerometer reads a single digit value,
249 typically 1-3°. when this value remains constant for 25 consecutive readings, it is assumed
250 that this is the range in which the anchor starts to move. The loadcell data within this range
251 is then assessed and an initial jump in the data can be seen and this indicates that the anchor
252 has moved. This point in time is then matched up with the initial movement shown in the
253 wireless gyroscope and accelerometer data and selected as the start point of the test.

254 Once the data has been synchronised, the data from the instrumentation can be converted
255 into the orientation of the anchor and the forerunner cable using the method outlined in
256 Robinson et al. (2019) with the pitch of the anchor and forerunner calculated using Equation

257 2. To determine the positional data of the anchor the 3 directions of acceleration recorded by
 258 the accelerometer, were corrected using the gyroscope data using the following equations:

$$Pitch = \tan^{-1} \left(\frac{A_x}{\sqrt{A_y^2 + A_z^2}} \right) \quad (2)$$

$$\theta = 0.98(\theta_a + \mu_a \Delta t) + 0.02\theta_a \quad (3)$$

259

260 where A_x , A_y and A_z are the accelerations recorded in the local x,y and z directions based on
 261 the orientation of the MEMs accelerometer, θ is the angle in a given direction, θ_a is the angle
 262 reported from the accelerometer, μ is the data from the gyroscope.

263 Once the accelerometer data had been corrected, the data was then passed through a Kalman
 264 filter using the following procedure:

$$X_k = \begin{bmatrix} x_k \\ y_k \\ \dot{x}_k \\ \dot{y}_k \end{bmatrix} = \begin{bmatrix} x_{k-1} + \dot{x}_{k-1}\Delta t + 1/2\ddot{x}_{k-1}\Delta t^2 \\ y_{k-1} + \dot{y}_{k-1}\Delta t + 1/2\ddot{y}_{k-1}\Delta t^2 \\ \dot{x}_{k-1} + \ddot{x}_{k-1}\Delta t \\ \dot{y}_{k-1} + \ddot{y}_{k-1}\Delta t \end{bmatrix} \quad (4)$$

265

266 Where X_k is the current state, x_k is the current position in the x direction, y_k is the current
 267 position in the y direction \dot{x}_{k-1} denotes the velocity of the previous timestep \ddot{x}_{k-1} is the
 268 acceleration in the x direction at the previous timestep. This can then be simplified as
 269 follows:

$$X_k = \begin{bmatrix} 1 & 0 & \Delta t & 0 \\ 0 & 1 & 0 & \Delta t \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} X_{k-1} + \begin{bmatrix} \frac{1}{2}(\Delta t)^2 & 0 \\ 0 & \frac{1}{2}(\Delta t)^2 \\ \Delta t & 0 \\ 0 & \Delta t \end{bmatrix} a_{k-1} \quad (5)$$

270

271 where X_k is the current state, X_{k-1} is the previous state and a_{k-1} is the vector of the previous
 272 acceleration in the x and y directions. A matrix A and B can then be fined as:

$$A = \begin{bmatrix} 1 & 0 & \Delta t & 0 \\ 0 & 1 & 0 & \Delta t \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \quad (6)$$

273

$$B = \begin{bmatrix} \frac{1}{2}(\Delta t)^2 & 0 \\ 0 & \frac{1}{2}(\Delta t)^2 \\ \Delta t & 0 \\ 0 & \Delta t \end{bmatrix} \quad (7)$$

274

275 The measurement model z_k can then be defined as:

$$z_k = HX_k + v_k \quad (8)$$

276

$$z_k = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_k \\ y_k \\ \dot{x}_k \\ \dot{y}_k \end{bmatrix} + v_k \quad (9)$$

277

278 where H is the transformation matrix, and v_k is the velocity vector of the current timestep.

279 The process convergence matrix (Q) for the 2-D Kalman filter can be defined as :

$$Q = \begin{bmatrix} x_k \\ y_k \\ \dot{x}_k \\ \dot{y}_k \end{bmatrix} \begin{bmatrix} S_x^2 & 0 & S_x S_{\dot{x}} & 0 \\ 0 & S_y^2 & 0 & S_y S_{\dot{y}} \\ S_x S_{\dot{x}} & 0 & S_{\dot{x}}^2 & 0 \\ 0 & S_y S_{\dot{y}} & 0 & S_{\dot{y}}^2 \end{bmatrix} \quad (10)$$

280

281 where S_x and $S_{\dot{x}}$ are the standard deviations of the position and velocity respectively, which
 282 can then be defined using the standard deviation of the measured acceleration using the
 283 following assumption:

$$Q = \begin{bmatrix} \frac{\Delta t^4}{4} & 0 & \frac{\Delta t^3}{2} & 0 \\ 0 & \frac{\Delta t^4}{4} & 0 & \frac{\Delta t^3}{2} \\ \frac{\Delta t^3}{2} & 0 & \Delta t^2 & 0 \\ 0 & \frac{\Delta t^3}{2} & 0 & \Delta t^2 \end{bmatrix} S_a^2 \quad (11)$$

284

285 where S_a is the magnitude of the standard deviation of the acceleration. The measurement
 286 noise covariance matrix R can then be defined as:

$$R = \begin{bmatrix} S_x^2 & 0 \\ 0 & S_y^2 \end{bmatrix} \quad (12)$$

287

288 Specifying the initial position of the anchor, a timestep of 20 ms (due to the 50Hz logging
 289 rate), and the corrected acceleration values in the x and y direction (based on the orientation
 290 of the anchor with respect to the gravitational field), the aforementioned matrices can be
 291 input into the KalmanFilter.py functions in python or the Kalman filter function available in
 292 Matlab, to obtain the current state X_k of the anchor with time (and by association drag
 293 distance). A flow chart outlining the steps taken to calculate the penetration depth can be
 294 seen in Figure 4.

295 4 Results and discussion

296 The following section presents the comparison of the data obtained from the 1g and
 297 centrifuge tests in all soil densities investigated. Results and dimensions are shown at
 298 prototype scale, unless otherwise stated. The results section is split into three sections, the
 299 first outlining the load displacement behaviour of the anchor, the second the attitude of the
 300 anchor during the anchor pull and third and final section discussing the reliability and
 301 experience using the developed instrumentation.

302 4.1 Scaling

303 The scaling laws adopted in this study for both the 1g and centrifuge testing are shown in
 304 Table 2 as summarised by Robinson et al. (2019).

305 *Table 3: Comparison of model scaling factors used for 1g and centrifuge testing (Robinson et al., 2019)*

Parameter	Scaling factor for 1g testing	Scaling factor for centrifuge testing
Acceleration	1	N
Length	1/N	1/N
Volume	1/N ³	1/N ³
Mass	1/N ³	1/N ³
Stress	1/N	1
Force	1/N ²	1/N ²

306

307 The adequacy of the 1g test scaling has previously been proven by Lauder and Brown (2014)
 308 that used a modelling of models technique that involved 1g testing of offshore pipeline
 309 ploughs at various scales (1/10th, 1/25th, 1/50th). This was later verified by Robinson et al.
 310 (2019) for dry and saturated testing of pipeline ploughs and Robinson et al. (2017) for cable
 311 ploughs by comparison to centrifuge testing and field tow forces from real installation
 312 campaigns. These verification processes have typically considered large deformation load-
 313 displacement behaviour and have not typically focused on plough kinematics and attitude.
 314 The same laws were adopted in this study for a buried drag embedment anchor.

315 4.2 Comparison of load-displacement behaviour of 1g and 316 centrifuge drag embedment anchor test 317

318 Figure 5 outlines the load-displacement behaviour of the 1g and centrifuge anchor drag tests
319 in all soil densities tested. The secondary axes of Figure 5 (top and right) have been
320 normalised by properties of the anchor, the drag distance has been normalised by the fluke
321 length and the tow force by the anchor weight. A comparison to a similar albeit more efficient
322 Stockless anchor has been added to Figure 5 utilising the design charts in the API RP 2SK
323 document (Petruska *et al.*, 2008). Two lines have been presented in Figure 5 to highlight the
324 25% reduction suggested for dense sands. The performance of the AC-14 anchor in the
325 experiments are in line with those of the Stockless anchor (a similar anchor), with a reduction
326 for dense sands also being present in Figure 5 for the AC-14 anchor tested although the
327 reduction is only 15%. From Figure 5 it can be seen that the load displacement behaviour of
328 the anchor is near identical when comparing the two testing methods with, both the stiffness
329 of the transition zone (distance to reach full depth and a steady state force) and the ultimate
330 holding capacity (*UHC*) being the same for a given density. When assessing the holding
331 capacity of the anchor it can be seen that for the loose ($D_r = 25\%$ and 38%) and the medium
332 dense ($D_r = 55\%$) soil beds the *UHC* value is 32% higher than in the dense soil bed ($D_r =$
333 82%). Although this may appear counter intuitive due to higher density soils having greater
334 resistance, this phenomenon can be explained by orientation of the anchor at *UHC* and the
335 penetration depth that is reached when the anchor reaches steady state.

336 From previous studies, and the results of the current investigation, it has been identified that
337 the penetration depth of the anchor is inversely proportional to the relative density of the
338 sand bed, with looser soils producing higher penetration than denser soils (Figure 6) (Naval
339 Civil Engineering Laboratory, 1982; Moore *et al.*, 2021). The lack of penetration results in

340 a lower overburden stress and as such a reduction in the resistance. The anchor will attempt
341 to reach a position in which the forces acting on the front and rear faces of the fluke are in
342 equilibrium and in general for fixed fluke anchors this is likely to result in the same value
343 for *UHC* for the majority of soil densities for a given anchor geometry. In the case of the
344 anchor used within this study, the geometry of the anchor changes as it is being pulled along,
345 as the anchor fluke is able to open to whatever angle it requires to meet equilibrium, as such
346 it is not necessarily the case that the opening angle of the anchor is the same for all soil
347 densities, which could result in different *UHC* values occurring in the different soil relative
348 densities. To investigate this the orientation of the anchor was assessed during the pull to
349 determine how the opening angle evolves with drag distance.

350 4.3 Comparison of orientation and embedment depth of anchor

351 In the case of this study the geometry of the anchor is able to change (Fluke opening angle,
352 Figure 2) during the dragging event as the opening angle of the AC-14 anchor is not fixed at
353 the start of the test (in both the 1g and centrifuge tests) (Figure 7). This results in the opening
354 angle of the anchor changing as the anchor opens up when being pulled and remains in a
355 certain position after the transition length.

356 Figure 7 shows the orientation of the anchor during the pull, with the fluke angle, shank
357 angle and the opening angle all outlined for all relative densities tested in the centrifuge, a
358 graphical representation of the anchor attitude at the end of the pull can be seen in **Error!**
359 **Reference source not found.** From Figure 7a-7c it can be seen that transition length of the
360 lower three densities is similar with the anchor reaching its equilibrium point (flattening of
361 the curve) by a distance of 8.0 m whereas in the dense soil bed this point is at approximately
362 5.0 m of horizontal displacement. The assumption of Equation 1 is that the maximum
363 opening angle is achieved by the anchor for all soil types and densities, it is also assumed

364 that the shank is near horizontal to the sand surface. From the data obtained from the anchor
365 tests it can be seen in Figure 7a that the shank angle (Figure 1b) varies significantly with
366 relative density. The range of shank angle to the horizontal is from 27° in the loose soil bed
367 ($D_r = 25\%$) to 5° in dense soil bed ($D_r = 83\%$) with no cases showing a horizontal shank
368 angle. In terms of the opening angle (Figure 1b), shown in Figure 7c, only the loose soil bed
369 ($D_r = 25\%$) nearly achieved the full opening angle of the anchor, at 31° with the opening
370 angle reducing as the relative density increases. In the case of the densest soil bed tested, the
371 opening angle is reduced to only 10° indicating that in this soil bed it was not possible for
372 the anchor to open, which may explain the differences between the *UHC* values from Figure
373 5.

374 Figure 7b shows that the angle of the fluke to the horizontal does not increase significantly
375 during the drag event with the angle to the horizontal ranging from 6 degree to 3 degrees at
376 the end of the anchor pull, fluctuations in the data are attributed to adjustments in the fluke
377 angle as the anchor is being dragged along to maintain an equilibrium of forces on the fluke
378 as there appears to be an optimum *UHC* that is maintained. The fluctuations only occur in
379 the loose soil and are thought to be due to fluctuations in soil bed density due to difficulty
380 in producing loose soil beds. The angle of the fluke has a small density dependency with
381 the looser soil showing a flatter angle of the fluke compared to the dense soil bed. This could
382 be attributed to a difficulty in the fluke penetrating into the soil as a larger resistance would
383 be met as the relative density increases. The results also show that the assumption of zero
384 fluke angle at *UHC* adopted in analytical (Neubecker and Randolph, 1996) approaches may
385 not be appropriate.

386 Figure 7c also shows that as the sand relative density decreases the opening angle of the
387 anchor increases, with the test in the loose soil bed reaching an opening close to the
388 maximum (35°) at 31° whereas in the dense soil bed the anchor only achieves an opening of

389 15° at the end of the anchor pull. The opening angle is predominantly determined by the
390 shank angle (Figure 7a) and can therefore be linked to the depth of penetration, as it assumed
391 that the forerunner cable (Towing cable) of the anchor is taught during the pull and inline
392 with the anchor shank due to the necessity of it being relatively short. To confirm the
393 assumption of the shank angle based upon forerunner inclination to the horizontal,
394 exhumation of the anchor was conducted after each pull, and the angle of the forerunner
395 cable and the shank of the anchor were manually recorded using a digital inclinometer and
396 compared. The physical measurements of the cable and shank matched and were within 0.1
397 degrees of the calculated values using the instrumentation. As the anchor penetrates deeper
398 the inclination of the forerunner cable increases, resulting in the anchor opening up further.

399 Using the instrumentation in the anchor and the Kalman Filter is possible to approximate
400 the depth of the anchor. Figure 6 shows the depth of all the anchors tested within this study.
401 An additional point has been added to each data set to verify the depth of the anchor after
402 the test had been conducted, with this data point being obtained by manual excavation of the
403 anchor and measurements to surface using callipers. From **Error! Reference source not**
404 **found.** it can be seen that as the relative density increases there is a decrease in the
405 penetration depth of the anchor, with the dense soil bed having a penetration depth of 0.7 m
406 or 0.4 fluke lengths and in the loose soil bed the anchor has a penetration depth of 4.2 m or
407 2.5 Fluke lengths. Comparing these values with the recommendation of penetration depth of
408 1 fluke length by Naval Civil Engineering Laboratory (1982) which is cited in the Carbon
409 Trust (2015) documentation it can be stated that for looser soils ($D_r < 40\%$) the anchor is
410 able to penetrate deeper than the recommendations. For the $D_r = 25\%$ and 32% the
411 penetration was 150% and 50% larger than the 1 Fluke length suggested by Nation Civil
412 Engineering Laboratories (1984) (as indicated by the “Hard soil” line on Figure 6) as also
413 shown in Sharif et al. (2023).

414 **Error! Reference source not found.** compares the penetration depths achieved in the 1g
415 and centrifuge model tests for all relative densities. From Figure 6 it can be seen that the 1g
416 testing is able to accurately recreate the results of the centrifuge tests in the medium dense
417 ($D_r = 55\%$) and the dense ($D_r = 82\%$) sand beds, but over predicts the penetration by a small
418 amount (6%) in the loose ($D_r = 25\%$ and 38%) sand beds. As the over prediction is small
419 and the depth is greater in the 1g tests, the ability of 1g testing to capture the penetration
420 depth of the anchor is thought to be adequate across all sand relative densities.

421 Figure 9a and 9b compare the attitude of the centrifuge and 1g anchor tests in the loose (D_r
422 = 25%) and dense ($D_r = 82\%$) respectively. From Figure 9 it can be seen that the small
423 changes in the attitude of the anchor exist between the 1g and centrifuge experiments, with
424 the difference in orientation being greater in the loose soil bed than in the dense, as was also
425 shown in the penetration depth data. The largest difference in Figure 9a is seen in the shank
426 angle, which is directly related to the penetration depth, the angle to the horizontal of the
427 anchor shank is lower in the centrifuge tests due to the reduced penetration depth. The
428 fluctuations in the loose soil data are thought to be a result of the difficulties in preparing
429 loose sand beds but make no overall significant difference to the kinematic behaviour of the
430 anchor. In the dense soil bed (Figure 9b), the data shows that the 1g test is able to reproduce
431 the findings of the centrifuge experiment with both the fluke and shank angles matching.
432 Overall, the centrifuge and 1g behaviour of the anchor is consistent in terms of both the
433 kinematic properties and the ultimate holding capacity. Therefore, the assumptions made for
434 ploughing proposed by Robinson *et al* (2019) seem to be extendable for DEAs of the scale
435 used within this study.

436 The reason behind the kinematic and behavioural similarities between the centrifuge and 1g
437 tests is down to the behaviour of large displacement of low effect stress boundary value
438 problems such as those experienced by pipeline (Lauder *et al.*, 2013) and cable ploughs

439 (Robinson *et al.*, 2019). As discussed by Neubecker and Randolph (1996) the soil resistance
440 and therefore the forces acting upon the anchor are generated by a series of consecutive
441 passive wedges, with a shear plane propagating from the tip of the fluke to the soil surface.
442 Lauder *et al.*, (2013) explained the approach behind why small scale 1g testing was able to
443 replicate field scale performance using a model of models approach for pipeline ploughs,
444 which have a similar soil failure mechanism. The friction angle along the shear plane evolves
445 and travels along the failure plane resulting in only relatively limited extents reflecting peak
446 behaviour. The length of the shear plane associated with the peak friction angle is
447 approximately $176 D_{50}$ according to Stone and Wood (1992) which would be 24 mm for the
448 HST95 sand used within this study. This equates to 12% of the shear plane at centrifuge
449 scale and 31% at 1g. The increase in soil resistance from the critical state portion of the shear
450 band are accounted for in the scaling laws, due to the additional N in the multiplication to
451 calculate force (force scaling from 1g is N^3 and for centrifuge is N^2). This accounts for the
452 reduction of effective stress attributed to the lack of an enhanced g field. The tip domain
453 (dilating front) reflects movement from peak to critical state behaviour and is at the leading
454 edge of the forming shear plane. Therefore, as the shear plane is at its full length the shear
455 stress associated with the tip domain is relatively small and therefore the enhanced dilation
456 does not influence the behaviour of the anchor significantly.

457 In terms of the effect on the trajectory of the anchor, as the influence of the dilating front is
458 minimal it is thought that the overall forces acting on the individual components of the
459 anchor are proportional in the 1g and centrifuge case (and the attitude is controlled by the
460 balance of forces). This means that the orientation of the anchor at any given point would
461 be similar for the centrifuge and 1g cases and the equilibrium point for the next horizontal
462 increment would follow a similar pattern.

463 Using the instrumentation outlined in this study has enabled the tracking of both the shank
464 and fluke of the anchor, which has not previously been possible using a non-contact
465 approach. This has allowed for the evolution of the anchor orientation and penetration to be
466 monitored over the length of the drag event. It was observed that the anchor does not open
467 to its full extent in all soil densities, contrary to the assumption of Equation 1, with looser
468 soils having a larger opening angle and denser soils having very narrow opening angle. It
469 was also found that the shank angle was dependent on the penetration depth of the anchor
470 and the relative density of the soil and not always near horizontal, although this is the case
471 for the medium dense and dense soil beds. This indicates that for looser soils Equation 1
472 would need to be modified to include, not only the opening angle of the anchor but also the
473 shank angle to the horizontal, as this will inform the direction and size of the soil wedge
474 formed when pulling the anchor across the seabed.

475 5 Conclusion

476 This paper has presented a comparison of the load-displacement and kinematic behaviour of
477 the 8.5 tonne AC-14 anchor in homogeneous sand beds of different relative densities tested
478 using both 1g and centrifuge model testing under drained sand conditions. The results of the
479 investigation have shown that as density decreases the penetration of the anchor increases,
480 and that the anchor penetration in loose soils is up to 2.5 times higher than previously
481 predicted. The experimental results represent anchors that have been pulled under drained
482 conditions using a cable rather than a chain, which is assumed to be the worst case scenario.
483 Further work is required to assess the influence of using a chain and on the influence of rate
484 effects on the anchor behaviour.

485 The 1-g model test was able to replicate the holding capacity of the anchor when scaled up
486 to full size using the scaling factors proposed by Bransby et al. (2005) for ploughing which
487 have previously been validated by Robinson *et al.* (2017) for all soil densities tested. The
488 kinematic behaviour of the anchor is comparable at 1g and centrifuge modelling, in terms
489 of its attitude during and at the end of a test. The penetrations depth of the anchor at the end
490 of a pull was shown to be similar in the 1g and centrifuge experiments with there only being
491 a 6% increase in depth in the loose ($D_r = 25\%$) at 1g when compared to its centrifuge
492 counterpart. Thus, for certain large strain problems 1g testing can be a reliable and valid
493 alternative to centrifuge testing even for low stress near surface problems.

494 The wireless instrumentation developed was shown to work well at transmitting data
495 wirelessly from the anchor to a logging PC and was able to measure the inclination of the
496 anchor in real time. Post excavation of the anchor showed that the instrumentation was able
497 to accurately measure the orientation of the anchor. Using the orientational data recorded
498 during the pull, it is possible to accurately calculate the penetration depth of the anchor, with
499 the post drag depth matching the calculated value. This study has shown that it is possible
500 to track very small complex buried moving objects in both 1g and centrifuge testing using
501 lows cost widely available logging and wireless communication systems using simple
502 techniques that can be easily implemented.

503 6 Acknowledgements

504 This work was funded through the UKRI EPSRC grant EP/W000954. W000997, W000970
505 “Offshore Cable Burial: How deep is deep enough?”, in collaboration with Durham
506 University and the British Geological Survey.

507

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582

583 **Figures**

584 Figure 1:a) Schematic diagram of a common AC-14 anchor used within this study
585 (dimensions can be seen in Table 1), b) Definition of anchor orientation

586 Figure 2: Image of 3D printed metal AC-14 anchor a) image of anchor as its placed on the
587 soil surface, b) image of anchor and wireless instrumentation (GB pound coin shown for
588 scale)

589 Figure 3: schematic diagram of centrifuge anchor installation apparatus utilised for both
590 centrifuge and 1g testing, a) assembly of apparatus and attachment of anchor to the towing
591 arm, b) diagram of complete actuator and strong box (soil is not shown for clarity)

592 Figure 4: Flow chart outlining the steps required to calculate the penetration depth of the
593 anchor from the recorded data

594 Figure 5: Comparison of the tow force vs drag distance for anchor tests conducted at 1g and
595 centrifuge model testing with comparison to a higher specification stockless anchor type in
596 API RP 2SK.

597 Figure 6: Comparison of the anchor penetration depth from 1g and centrifuge model
598 anchor tests (open symbols are measured at the end of test to verify depth). Comparison

599 also with previous drum centrifuge testing (Moore et al 2021) and the field CBRA
600 approach (Carbon Trust, 2015)

601 Figure 7: Orientation of the model anchor during test, angles taken from the horizontal
602 axis, a) Shank angle, b) fluke angle, c) opening angle (all data in this figure is from
603 centrifuge tests)

604 Figure 8: Schematic representation of the orientation of the anchor at the end of the pull
605 (note that the anchor at not shown at the final depth but are all displayed as if at the
606 surface)

607 Figure 9: Comparison of anchor attitude in 1g and centrifuge tests within a given soil
608 density, a) loose soil bed ($D_r = 25\%$), b) dense soil bed ($D_r = 82\%$)

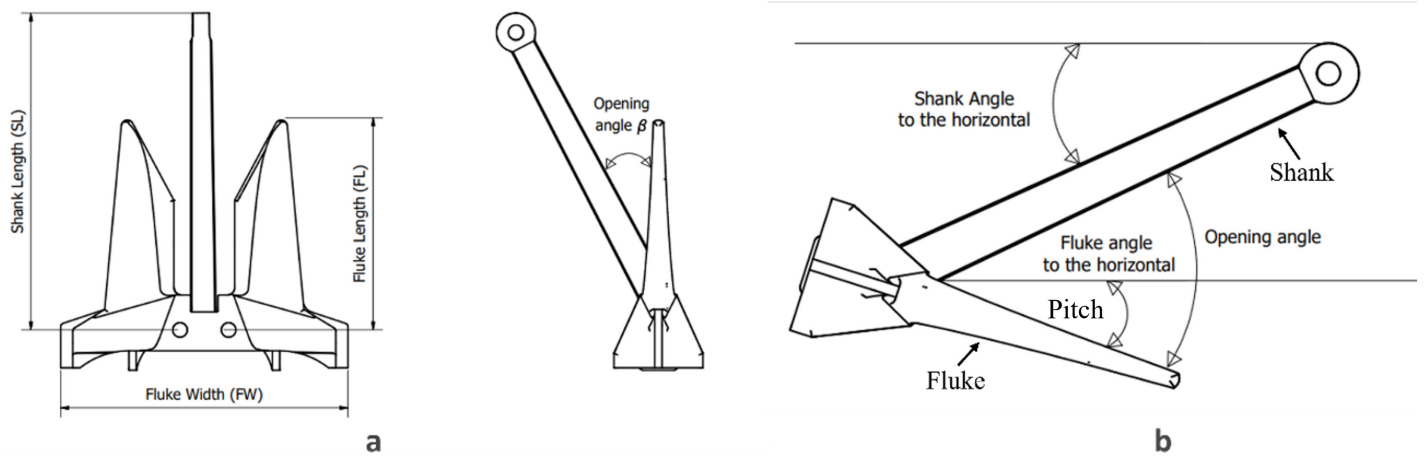
609 Tables

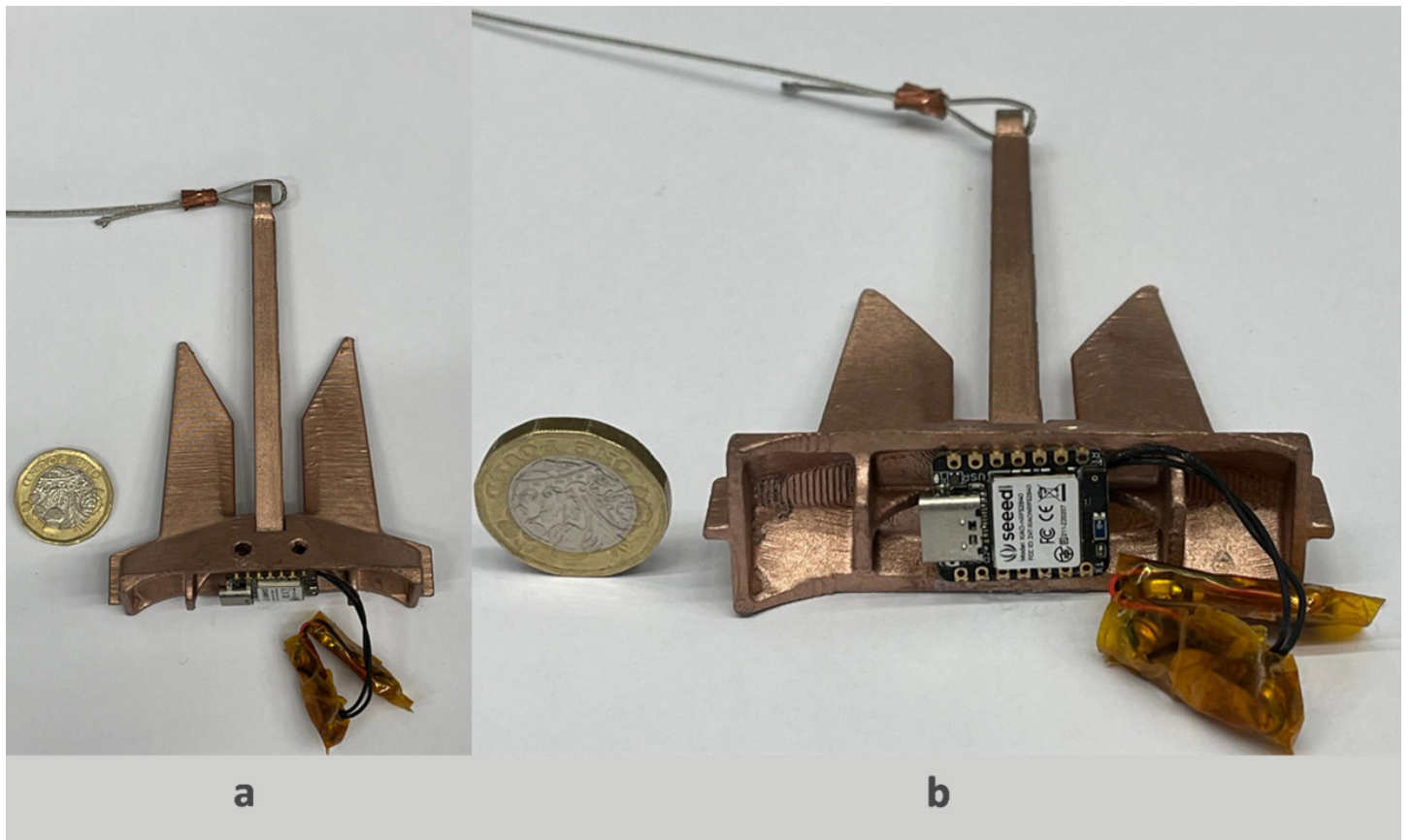
610 Table 1: HST95 sand material properties (Al-Defae et al, 2013; Lauder et al., 2013)

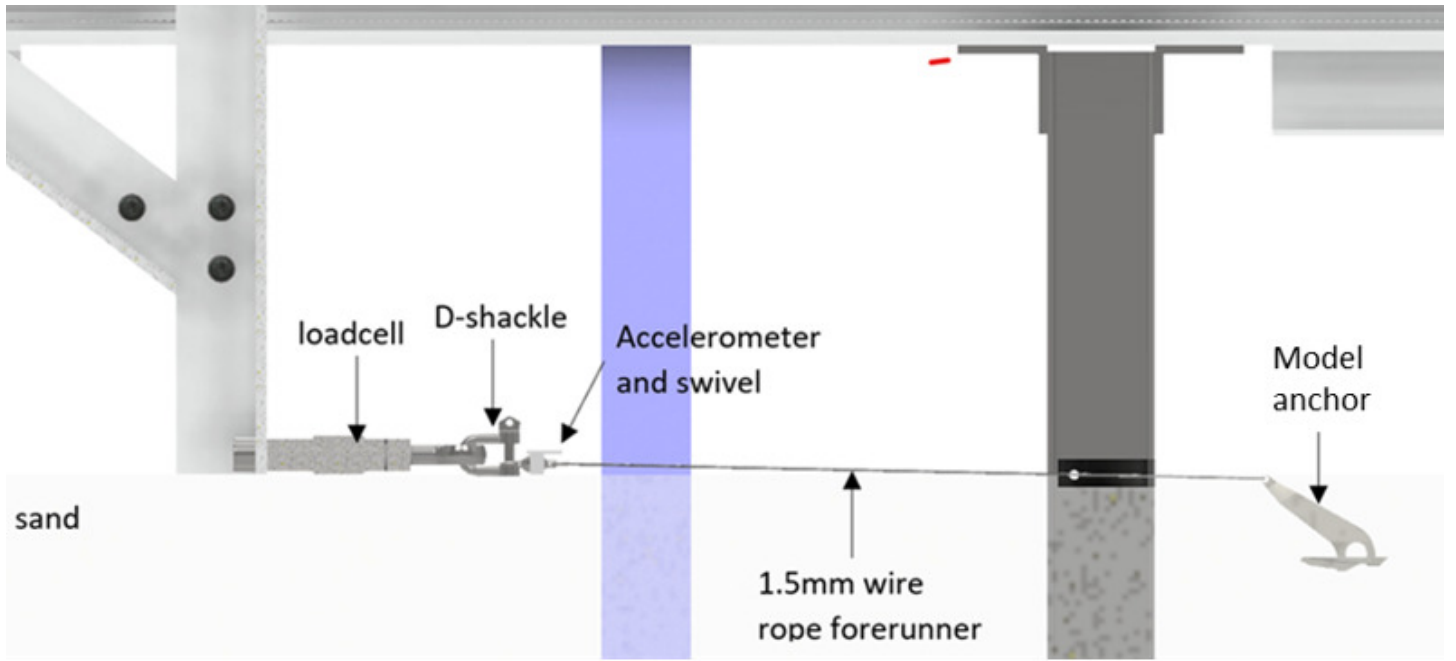
611 Table 2: Properties of the model and prototype AC14 anchor geometry

612 Table 3: Comparison of model scaling factors used for 1g and centrifuge testing (Robinson
613 et al., 2019)

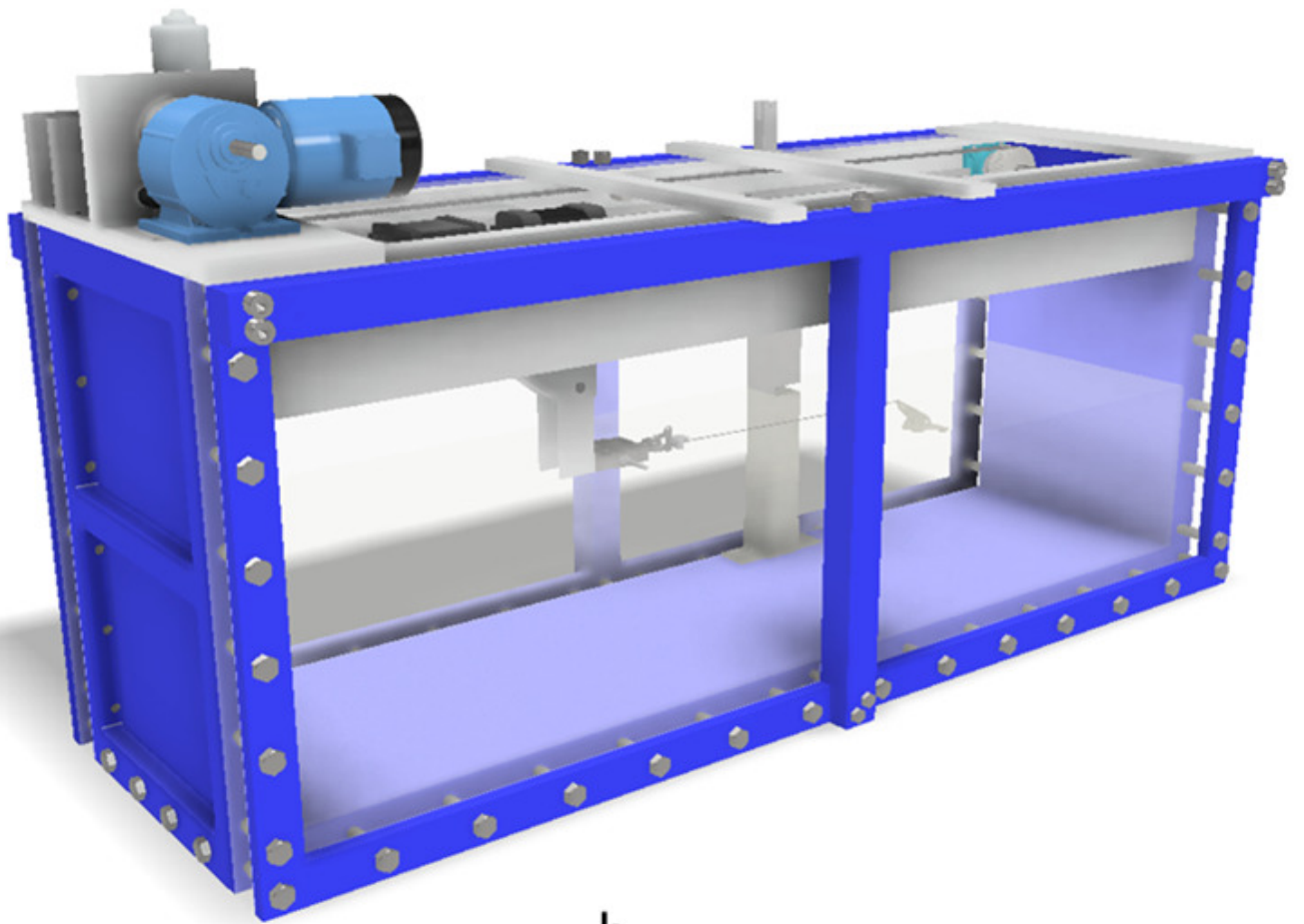
614







a



b

